ENERGY CONSERVATION IN COAL CONVERSION

Energy Conservation Potential in Heat Recovery Techniques

A Case Study

J. D. Stas

Carnegie-Mellon University Pittsburgh, PA 15213

June, 1978

Prepared for

THE U.S. DEPARTMENT OF ENERGY Pittsburgh Energy Technology Center UNDER CONTRACT NO. EY775024196

ABSTRACT

In this study, we looked at replacing certain heat exchangers with Organic Rankine Cycles. In each case, we determined the cost of generating power and then from this tabulation of capital investment for power generation, feasibility of replacement on a unit-by-unit basis was determined.

The results show that 18 heat exchangers reject sufficient heat to warrant ORC-usage, with potential electric generation of 36 MW or a 17% increase of the inplant power generation of 210 MW.

Cost estimates indicate the capital investment required to be approximately \$1000/KW with a potential reduction of \$300/KW for mass produced units.

Based on the results of this analysis it is recommended that ORC manufacturers be engaged to further engineer and incorporate Organic Rankine Cycles into the Oil/Gas design.

TABLE OF CONTENTS

	Page
1.	Introduction
2	W. 5 to test the first test test test test test test test t
2.	H _x Suitability for ORC
3.	Organic Rankine Cycle
4.	Cost Analysis
	4.1 DCF Sensitivity Analysis
,-	4.1 DCF Sensitivity Analysis III-28
5.	Discussion of ORC III-29
6.	Conclusions
7.	Recommendations
0	Recommendations
8.	References
	Appendix A - Sample Calculations
	Rankine Cycle Power Output
	Appendix B 1. Cost Analysis III-37
	2. DCF - Sensitivity Analysis III-38

III-4

LIST OF TABLES

		Page
Table 1. Output	t Potential-AFI Energy Systems	III-15
. —	Output-Barber and Nichols	III-16
	ry of ORC Analysis	III-19
	Involved with ORC Applications	
	LIST OF FIGURES	
		Page
Figure 1.	Comparison of ORC and H_X	III- 6
Figure 2a.	Present Air and Water Cooled Design	III- 7
Figure 2b.	Alternate Rankine Cycle Design	111- 7
Figure 3.	500 KW Demonstration Unit	III- 9
-	3,800 KW Commercial Unit	111-10
Figure 4.	Output from Liquid Heat Source	III-ll
Figure 5.	Output from Condensable Heat Source	
Figure 6.	Rankine Cycle Efficiency vs. Maximum Cycle	
Figure 7.	Temperature	111-14
Figure 8.	Estimated Installed Costs for Rankine Cycles	III-18
Figure 9.	Sensitivity Curves @ 2.5¢/KWH	III-40
Figure 10.	Sensitivity Curves @ l¢/KWH	8

1. Introduction

In our initial energy study, we developed a number of methods by which energy can be conserved in inefficient coal conversion plants. 10 Currently, our objective is to apply the procedures we have learned to more near term, efficient and highly engineered plants. The commercial concept 0i1/Gas Complex designed by Ralph M. Parsons Company has been selected as the next candidate to be evaluated. This design has a high thermal efficiency of 77%. 11

The purpose of this particular study is to investigate the feasibility of replacing certain heat exchangers with an organic rankine cycle. For each case, the cost of generating electric power is to be determined and then from this tabulation of capital investment for power generation, the feasibility of replacement on a unit-by-unit basis will be determined.

2. H_X Suitability for ORC

Every heat exchanger in the Oil/Gas Complex has been evaluated for its suitability of being replaced by an organic rankine cycle to produce shaft work. As shown in Figure 1, the ORC can perform essentially the same function as a heat exchanger but the exit temperature of the second stream cannot be the same, $(T_4 \neq T_4^{-1})$ since work is extracted.

In evaluating heat exchangers, there are three reasons why a heat exchanger may be rejected as a potential candidate: (1) the exchanger's operation is important to the downstream process and therefore a temperature change in any stream cannot be afforded or (2) the incoming temperature of the process stream is too low to warrant ORC usage or (3) the unit is too small.

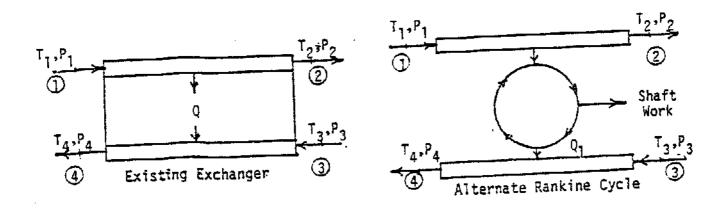


FIGURE 1. COMPARISON OF ORC AND ${\rm H}_{\rm X}$

3. Organic Rankine Cycle

In many areas in the Oil/Gas complex, air coolers and water coolers are used to cool process streams. In some cases the coolers are used independently and in others they are used in series as shown in Figure 2A. Normally, the air cooler cools the stream to 120°F, then the water cooler cools the stream to 100°F. The inlet temperature of the air coolers vary throughout the plant from 550°F-200°F. It is these schemes which are proposed for replacement by the Rankine Cycle design in Figure 2B, in this report.

working fluid to produce shaft power through a reciprocating or turbine type expander. The air cooler and/or water cooler is replaced by the boiler of Figure 2B keeping inlet and exit states of the process stream constant. Therefore, the waste heat which was previously lost to the atmosphere is used as a heat source for the Rankine Cycle in which some of this heat is converted to mechanical energy in the expander while the remaining is rejected in the condenser to the cooling tower.

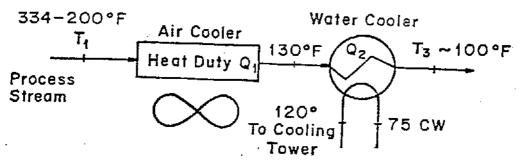


FIGURE ZA. PRESENT DESIGN

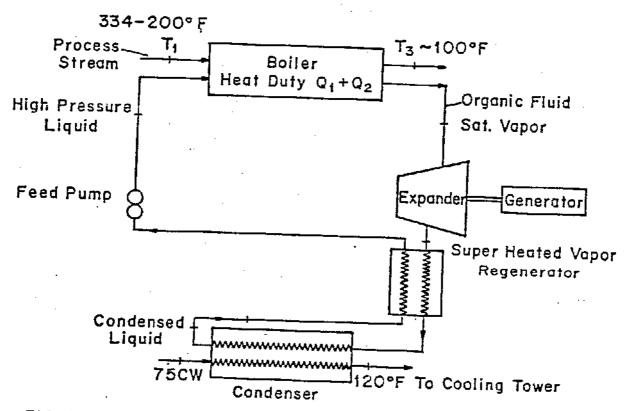


FIGURE 28. ALTERNATIVE DESIGN ORGANIC RANKINE CYCLE

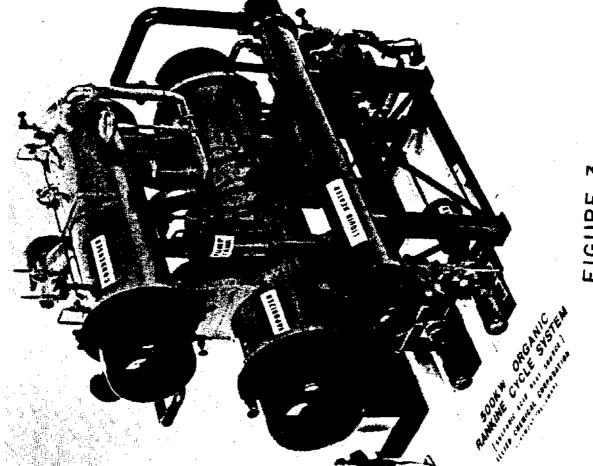
To date, only AFI Energy Systems is in a position to market Organic Rankine Cycles as low level waste heat recovery systems in the 200-400°F temperature range. AFI Energy Systems is a joint venture of: Allied Chemical, Foster Wheeler and Ishikawajima Hauma (IHI) of Japan.

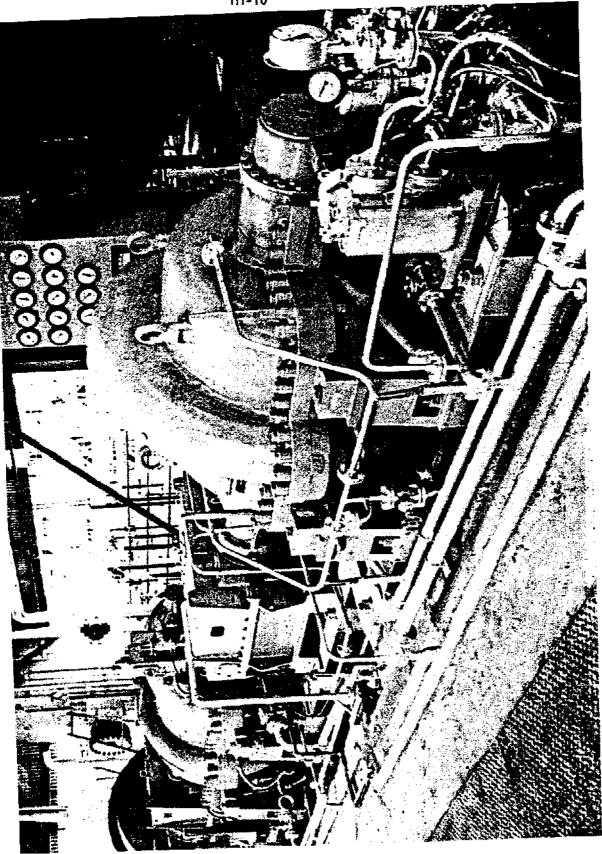
A demonstration 500 KW Organic Rankine Cycle system shown in Figure 3 is being constructed at the Allied Chemical facility at Claymont, Delaware and will be operating in early 1978. This plant incorporates a turbine and associated technology which has been commercially applied in Japan since 1968 in a 3800 KW Organic Rankine Cycle. This system shown in Figure 4 has provided over 70,000 hours of continuous operation with no major problems. 18

The AFI systems are being offered for sale on a turnkey, fixed price basis in four nominal sizes: 500 kW, 1000 kW, 2000 kW, and 4000 kW. Delivered costs are approximately \$1000/kW.

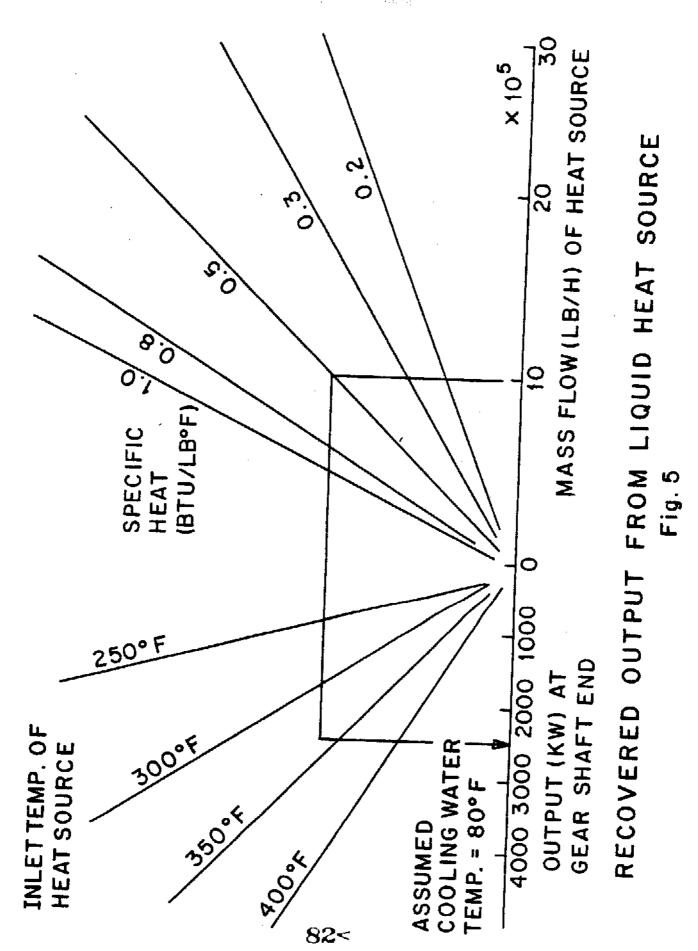
AFI's current market thrust is toward retrofitting the ORC to waste heat sources in existing plants. These systems utilize liquids or condensable vapors as a heating source. AFI feels economics are not yet justified for installation of an ORC when using gas as a heating source because of a much larger heat transfer area required in the boiler. A three-year AFI study indicated that there is a huge potential application in the following areas: chemical plants, refineries, chemical processes, and areas where there is excess process steam. 17

Figures 5 and 6, furnished by AFI, have been used to estimate the ORC power output potential of heat exchangers with a liquid or condensable vapor as a heat source. Samples are shown in the figures.

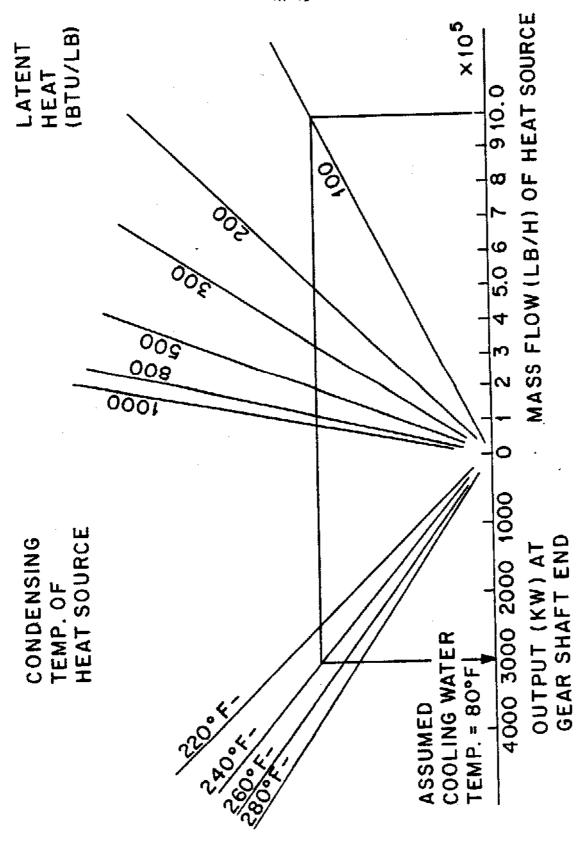




ORGANIC RANKINE CYCLE SYSTEM FIGURE 4 3800KW







RECOVERED OUTPUT FROM CONDENSABLE HEAT SOURCE Fig. 6

Table I shows the results of the power estimates. As can be seen in the table, this analysis indicates that 6 MW of power can be generated using AFI's hardware.

Our analysis of all heat exchangers in the Oil/Gas Complex shows that most of the rejected waste heat is removed from gas streams. Currently AFI does not market systems which can use this heat source, but this is because their market thrust is toward retrofitting in existing installations rather than application at the design stage of a new plant. When looking at the economics of the ORC in a new design, credit must be taken for the heat exchanger which would have otherwise been needed to remove the heat. This credit will make the ORC utilizing a gas heat source economically attractive.

Barber-Nichols Engineering Company (Refs. 6 and 7) has constructed a generalized curve showing the evaluation of Rankine Cycle efficiency with maximum cycle temperature for various working fluids as shown in Figure 7. It is on this curve that output power has been made for the ORC system utilizing a gas as a heat source. Sample calculations are given in Appendix A. All results are shown in Table II. The results of Table II show that by incorporating an ORC in every potential gas stream over 30 megawatts of power can be generated.

4. Cost Analysis

Only a rough figure of \$1000/KW for the ORC systems has been obtained through personal conversations with a representative of Allied Chemical. Installation costs have been kept at a minimum because of a modular installation approach and is estimated to be 20% of the capital

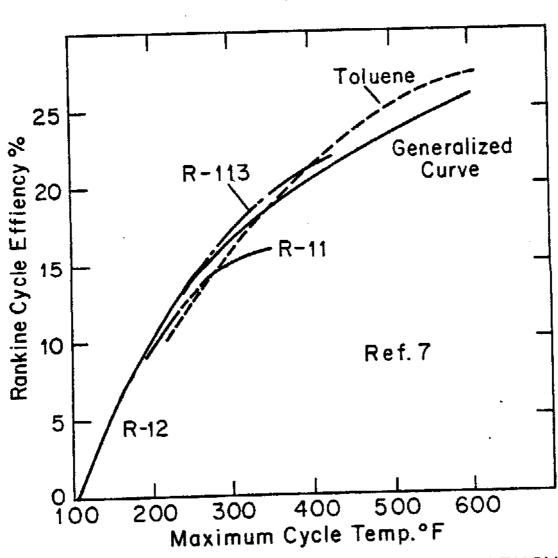


FIGURE 7. RANKINE CYCLE EFFICENCY

910/KW

Or Installed Cost of --

\$5,716,500

\$1,843,500

Heat Exchanger Costs --

Net Investment

\$7,560,000

Table I - Output Potential-AFI Energy Systems

Costs	£ 600,000	2,280,000	000*009	1,080,000	720,000	1,680,000	000*009	\$7,560,000
\$/KW	1200	1200	1200	1200	1200	1200	1200	1200
Output KW	200	1900	500	006	900	1400	200	6300 KW
Temp. of Source	450	260	300	460	280	300	270	T0TAL
Mass Flow 1b/hr	46,340	44,500	224,740	74,000	199,600	942,413	106,500	
Specific Heat BTU/1b-°F	0.45	0.45	0.41	0.65	0.65	0.45	0.57	
Description	slurry vapor condenser air	slurry vapor Water condenser	Hp separator liquid coolers	dried vapor cooler	naptha air cooler	fuel oil air cooler	product coolers	
Item	12-1301	12-1302	12-1307(8)	13-1301	55 14-1308	14-1314	16-1307(8)	

	Table II - Power Output-Barber and Nichols	Output-Barber	and Nichols	
Item	Description	Mass Flow 1b/hr	Temp. of Source	Power Generated KW
12~1305	Hp separator vapor air cond.	605,700	300	5400
16-1303	effluent air cooler	325,350	280	006
17-1304	amine cond,	153,500	230	2500
18-1303	methanation comp. lst stage dis- charge	294,300	290	1300
18-1304	methanation comp. 2nd stage inter- cooler	294,300	250	006
18-1308	methanation effluent air cooler	305,314	305	2200
18-1315	SNG comp. 1st stage discharge intercooler	277,295	235	800
21-1302(3)	shift gas coolers	2,019,162	560	8200
24-1307(8)	fuel gas coolers	2,213,382	300	7900
			101	TOTAL 30,100

investment. The installed cost is therefore approximately \$1200/KW for systems utilizing a liquid or condensable vapor heat source. The capital investment required, shown in Table I, is 7.5 million dollars. The total replacement heat exchanger cost was found to be 1.8 million dollars. Taking the heat exchanger costs as a savings the net investment is 5.7 million dollars or \$910/KW. It is assumed that the same cost will be realized with ORC's utilizing gas as a heat source when credit is taken for the replacement heat exchangers.

Figure 8 presents cost curves which were extrapolated from cost curves given in Reference 6. These curves forecast the installed costs of Rankine Cycles for production units. The costs for the Rankine system include all the components necessary to produce shaft power and, in addition, the generator and associated controls to result in electrical power generation. The additional cost to the cooling tower because of larger cooling requirements are not given in this figure.

These curves assume a 100% installation cost and a 6% escalating rate from 1976. From this figure, the installed cost of replacement Rankine Cycles was estimated based on the cycle output and maximum cycle temperature. The cost estimates are given in Table III. Sample calculations are given in Appendix B of the report.

Table III is a summary of the results of the ORC feasibility analysis for all heat exchangers in the Oil/Gas Design. This table gives the feasibility of replacement in column one and the type of feasible exchangers in column two. Column three gives the estimated power output of each replacement cycle. The estimated installed costs are given in columns four and five, column 4 is AFI's estimated costs and column 5 is

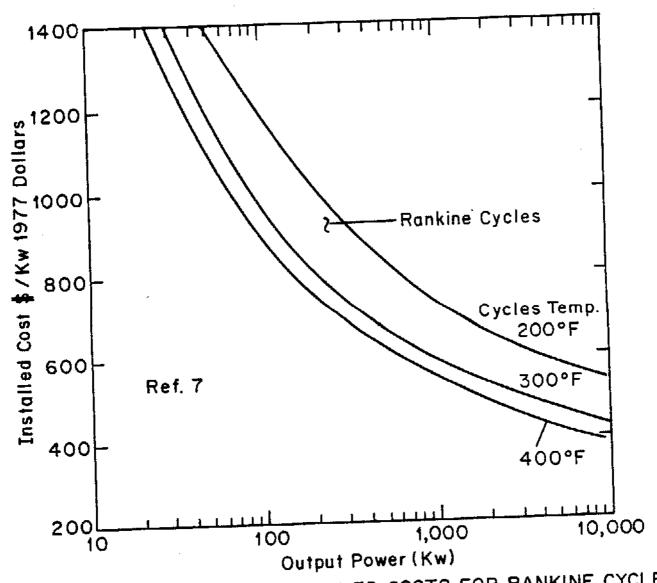


FIGURE 8. ESTIMATED INSTALLED COSTS FOR RANKINE CYCLES

Table III - Summary of ORC Analysis

	Description	Feasibility	Type	Power Generated KW	Installed Costs AFI (\$/KW) B & N	ts N Remarks
slurry vapor air condenser			steam-air	200	1200 725	· ·
slurry vapor Water condenser			steam-air	1900	1200 650	0
Hp separator slurry feed exchangers		rejected				Process Hx
		rejected				Process Hv
Hp separator slurry OH cond. Hx		rejected				Process Hx
Hp separator slurry steam gen.		rejected				Process HX
Hp separator vapor Feed Gas Hx		rejected				Process Hx
Hp separator OH vapor steam gen.		rejected				Process Hx
Hp separator OH vapor air condenser			gas-æir	5400	200	_
Hp separator OH liq. steam gen.		rejected				Process #x

90<

Table III - Summary of ORC Analysis

Remarks	Process Hx	+12-1307	Temp. too low	Process Hx	Output too low	Output too low	Process Hx	Output too low	Process Hx	Output too low	Output too low	Output too low
Installed Costs AFI (\$/KW) B & N	625											
Instal AFI (\$/	1200											
Power Generated KN	200											
Type	Liquid-air											
Feasibility			rejected	rejected	rejected	rejected	rejected	rejected	rejected	rejected	rejected	rejected
Description	Hp separator OH liq. air cooler	Hp separator OH liq. water cooler	Hp separator OH Cond. $ m H_2O$ cooler	Hp flash vapor steam generator	Hp flash vapor air condenser	Hp flash vapor water condenser	lst IP flash vapor steam generator	lst IP flash vapor air condenser	2nd IP flash vapor steam generator	2nd IP flash vapor air condenser	LP flash vapor air condenser	LP vent gas condenser
Item	12-1307	12-1308	12-1309	12-1315	12-1316	12-1317	A 12-1318	12-1319	12-1320	12-1351	12-1322	12-1324

Analysis
ORC
0£
Summary
- 111
Table

Remarks		Process Hx		Process Hx	Process Hx Process Hx	Process Hx	Process Hx	Process Hx	Process Hx	Process #x	Process #x	Process Hx	
Installed Costs AFI (\$/KW) B & N		280									_		675
Power Generated Ir	9001		•										1200
Type	liquid-air c												liquid air 600
Feasibility	,	rejected	rejected	rejected	rejected	rejected	rejected	rejected	rejected	rejected	rejected	rejected	liq
Description	Dried vapor cooler	Recycle wash oil preheater	Orier WHB #1	Drier WHB #2	Hy. Dist. PA/ Lt. Dist. Reboiler	Hy. Dist. PA Feed Exchanger	Hy. Dist. PA Steam generator	Fract. OVHD/ Steam generator	Lt. Dist. PA Steam generator	Fract. Bottoms/ Hy. Dist. reboiler	Main Fact. charge furnace	Fract. Bottoms/ Feed exchanger	NAPHTHA air cooler
Item	13-1301	13-1302	13-1601	13-1602	14-1305	14-1301	14-1302	14-1307	14-1303	14-1306	14-1401	14-1304	14-1308

Table III - Summary of ORC Analysis

			!	Power Generated	Installed Costs	8emarks
Item	Description	Feasibility	Type	X	AFE (\$/ NM / D & H	
14-1315	ATM. Bottoms/ 600 psig steam gen.	rejected				Process HX
14-1312	Fuel oil/150 psig steam generator	rejected				Process Hx
14-1313	Fuel oil/150 psig steam generator	rejected				Process Hx
14-1314	Fuel oil air cooler		liquid-air	1400	1200 590	
14-1309	OVHD. vapor interm. air cooler	rejected				Temp. too low
14-1310	OVHD. vapor air cooler	rejected				Temp. too low
16-1302	Charge heater	rejected				Process Hx
16-1301	Feed-effluent exchanger	rejected				Process Hx
16-1303	Effluent air cooler		gas-air	906	059	•
16-1304	Stabilizer feed- bottoms exchanger	rejected				Process HX
16-1307	Product air cooler		liquid-air	200	1200 725	4 1 1
16-1304	Stabilizer OVHD air cooler	rejected				Output too tow
16-1308	Product water trim cooler		liquid-H ₂ 0			+16-130/

Table III - Summary of ORC Analysis

								II	I-2:	3					19.
Remarks		Process Hx	Process Hx	Output too low	Process Hx	Process Hx	Process Hx	ı	Process Hx	Process Hx	Process Hx	Temp. too law		Process Hx	Process Hx
Installed Costs AFI (\$/KW) B & N								0				950	017		
Power Generated KW							2500					1300	006		
Type							gas-air	ı				gas-H ₂ 0	gas-H ₂ 0		
Feasibility	rejected	rejected	rejected	rejected	rejected	rejected		rejected	rejected	rejected	rejected	. בל בנ		rejected	rejected
Description	Stabilizer reboiler	Feed effluent Hx	Effluent cooler	Gas/Gas Hx	Amine cooler	Amine exchanger	Anine condenser	Amine reboiler	Amine reclaimer	Regeneration heater	Regeneration conler	Nethanation comp.	Methanation comp. 2nd stage discharge intercooler	Methanation feed/effluent fix	Methanation Start-up heater
Item	16-1306	19-1301	19-1302	17-1301	17-1302	17-1303	17-1304	17-1305	17-1306	18-1301	18-1302	18-1303	18-1304	18-1305	18-1401

Remarks	Process Hx		Process Hx	Frocess nx	Output too low 2-		Output too low	Output too low	Temp. too low	Process Hx	Output too low	Process Hx
	Proc		Proc	roc '	Out		Out	0 et	Ten	Pro	ort Ort	P.
Installed Costs AFI (\$/KW) B & N		570				780						
Power Generated		2200										
Туре		gas-air				gas-H ₂ 0						
Feasibility	rejected		rejected	rejected	rejected		rejected	rejected	rejected	r rejected	rejected	rejected
Description	Methanation circulating oil boiler	Methanation effluent air cooler	Polish methanator feed/effluent Hx	Polish methanator start-up heater	Polish methanator air cooler	SNG compressor 1st stage discharge intercooler	SNG comp. 2nd stage discharge cooler	Deethanizer comp. Ist stage discharge intercooler	Deethanizer cond.	Deethanizer reboiler rejected	Depropanizer cond.	Depropanizer re- boiler
Item	18-1306	18-1308	18-1311	18-1402	18-1313	18-1315	18-1316	18-1314	18-1317	18-1318	18-1319	18-1320

Table III - Summary of ORC Analysis

		z			-		III-2:	5			-		
Remarks	Tomp +po]	Process Hx	G	Process HX	Temp. too low	: :	Process HX Process HX	Process Hx	Process Hx	Process Hx		+21-1302	Temp. too low
Installed Costs AFI (\$/KW) B & N											520		
Power Generated KW											8200		
Type											gas-air	gas-H ₂ 0	
Feasibility	rejected	rejected	rejected	rejected	rejected	rejected	rejected	rejected	rejected	rejected			rejected
Description	Debutanizer cond.	Debutanizer re- boiler	Steam superheater	Oxygen preheater	Quench water air cooler	Steam boiler	170 psia waste heat boiler	Boiler feed water preheater	40 psia waste heat boiler	25 psia waste heat boiler	Shift gas air cooler	Shift gas water trim cooler	Quench Water air cooler
Item	18-1321	18-1322	20-1301	20-1302	20-1303	36 36	7 21-1601	21-1301	21-1602	21-1603	21-1302	21-1303	24-1301

Table III - Summary of ORC Analysis

Process Hx Process Hx Process Hx Process Hx
rejected rejected rejected rejected
Air/fuel gas HX #1 Fuel gas-1200 psi steam generator Air/fuel gas HX #2 Fuel gas-150 psi steam generator Fuel gas-25 psi

₹ III-27

V
75
Analysis
<u> </u>
2
₹
\mathfrak{S}
<u></u>
of ORC
_
>
Ξ.
ĕ
Summary
۲"
1
_
\blacksquare
-
انه
<u></u> !
Table III
끧
•

					_		- 1 <u>o</u>	Ш	-27	
Remarks	Process Hx	Process Hx	Process Hx	Process Hx	Temp, too low	Temp. too low	Potential too low			
Installed Costs AFI (\$/KW) B & N								260	230	300/KW
								1200	910	985/KW
Power Generated KW								36,400	-12,040,260	+ 2,300,000
Type								Total	Hx Investment	ng Tower Costs
Feasibility	rejected	rejected	rejected	rejected	rejected	rejected	rejected		Less Hx I	Plus Cooling To
Description	Condenser	Condenser	Condenser	Condenser	Condenser	Condenser	LP flash vapor air condenser	HX		
Item	32-1311	32-1312	32-1313	32-1315	32-1316	32-1317	8 12-1323 √	Total 110 Hx	18-Used	

the estimated costs from Reference 7. The last column gives reasons for rejection of heat exchanger replacement.

The total results shown in page 27 of the table indicates that 36 megawatts of power can be generated. The costs for the ORC is estimated to be around \$1200/KW using AFI's data and \$560/KW using the data from Reference 7. When credit is taken for the replaced heat exchangers and an adjustment made for the increased cooling tower costs the AFI estimate drops to \$985/KW and \$300/KW for Reference 7 costs.

Although the AFI estimates indicate the current costs of ORC for waste heat utilization, the costs from Reference 7 indicate the potential costs of the ORC given the appropriate demand. Given this range it is therefore necessary to perform a return on investment sensitivity analysis to demonstrate the potential ROI for various investment costs and selling prices.

4.1 DCF Sensitivity Analysis

A discounted cash flow analysis has been performed on varying sizes of ORC for different investment costs and electricity exporting rates and the results are shown in Figures 9 and 10. Figure 9 assumes a \$.025/KW-hr exporting rate escalating 8% per year for 10 years. Figure 10 assumes a \$.01/KW-hr exporting rate escalating 6% per year for 10 years. Assumptions used for the basis of this analysis are in accordance with the Gas Cost Guidelines used in the Oil/Gas Complex and are shown in Appendix B.11

The cost curves of Figure 8 were used as a basis for this analysis. The capital investment was taken directly from Figure 8 for curve B in Figure 9 (B and C), the most optimistic curve. The capital investment for the pessimistic outlook, curves (A and D) was assumed to be a 100%

increase in the curve of Figure 7. This analysis does take credit for replacement heat exchangers.

The expected ROI for two ORC manufacturers is also given in the figure. One is AFI at \$985/KW and the other is Sundstrand Corporation, a 600 KW waste heat recovery ORC utilizing heat source temperatures above 550°F. The Sundstrand systems installed cost is \$800/KW, with a mass production projection of \$400/KW.

5. Discussion of ORC

The results and conclusions presented here concerning Organic Rankine Cycles are not necessarily (restricted) to coal conversion plants but can be expanded to any inudstry in which low level heat is being wasted.

Rankine Cycles, waste heat can be utilized to produce useful electrical or shaft power. All ORC presented in this report are within the realm of technological development of Rankine Cycles. In addition to AFI Energy Systems and Sundstrand's experience, many other U.S. firms have applied considerable effort to the development of Organic Rankine Cycles for various applications. Table IV, not intended to be an all inclusive list, gives a summary of some of the companies working on ORC.

Most applications of the Organic Rankine Cycle are of a prototype nature at the present time and therefore costs are substantially higher than the estimates presented here. In some cases the costs are as high as \$2000/KW-\$3000/KW, but all manufacturers forecast price declines given the appropriate demand. AFI's and Sundstrand's cycles

Table IV

		Manufacturer	Type of Fluid	Type of Expander	Type of Application	Expander Inlet °F/PSIA	Rated Power Hp
		Aerojet-Liquid Rocket	AEF-78	Turbine	Automobile	020/1000	74.9
	۲,	Barber-Nichols	R-113	Turbine	Solar Cooling	200/57	2.7
	က်	Barber-Nichols	R-113	Turbine	Solar Irrigation	920/221	25.0
	4.	Fairchild-Hiller	FC-75	Turbine	Total Energy Plant	428/206	25.34
	5.	Kinetics	R-113	Rotary	Automobile	375/355	47.0
	6.	Kinetics	R-114	Rotary	Solar Cooling	200/180	7.5
	7.	Ormat	MCB	Turbine	Power Pack	variable	3.0
	8	Sundstrand Aviation	CP-25	Turbine	Total Energy Plant	825/195	134.1
10	9.	Sundstrand Aviation	Dowtherm A	Turbine	Power Pack	7/00/	8.0
1<	10.	Sundstrand Aviation	Tolvene	Turbine	Waste Heat Recovery	550/300	006
	Ξ.	Thermo-Electron	Fluorinol 85	Turbine	Automobile	001/009	145.5
	12.	Thermo-Electron	Fluorinol 85	Turbine	Gas turbine bottoming plant	600/100	
	3.	Thermo-Electron	Fluorinol 85	Turbine	Diesel engine bottoming plant	00//009	1,000
	14.	United Aircraft	R-113	Turbine	Solar Cooling	200-375/70-340	4.3
	15.	United Aircraft	R-114	Turbine	Solar Cooling	250-275/250-400	8.0

are currently being sold at reasonable costs with satisfactory rates of return given today's electricity costs.

These efforts and the efforts of numerous other companies indicate the cost estimates presented here are certainly within the time frame necessary for use in coal conversion plants.

6. Conclusions

Based on estimates and results presented in this report, the following conclusions are drawn:

- The Organic Rankine Cycle is an energy effective
 to air and water cooled systems operating at
 temperatures above 200°F. In the Oil/Gas
 Complex 36 megawatts of electricity can be
 produced in an energy effective manner through
 recovery of the waste heat of air and water
 coolers.
- 2. Incorporating Organic Rankine Cycles into coal gasification designs will generate demand to lower production costs and, therefore, enable the ORC to become cost effective in a variety of other industries where waste heat is available.

 On a national level the energy savings potential is incredible.

7. Recommendations

This report is a preliminary analysis which pinpoints 18 heat exchangers throughout the Oil/Gas Complex, in which the rejected heat is sufficient to generate over 36 MW of power via Organic Rankine Cycles. It is therefore recommended that current manufacturers be contacted and steps taken to further engineer and incorporate Organic Rankine Cycles into the Oil/Gas design.

8. References

- Van Wylen, Sonntag, "Fundamentals of Classical Thermodynamics", 2nd edition, John Wiley and Sons, Inc., New York, 1973.
- Gagiolo, Pitit, "Second Law Analysis, for Pinpointing the True Inefficiencies in Fuel Conversion Systems", Mechanical Engineering Dept., Marquette University, Milwaukee, Wisconsin 53233.
- Morgan, D. T., Davis, V. P., Newton, C. L. and Stonehocker, V. T., "High Efficiency Diesel/Organic Rankine Cycle Combined Power Plant", ASME 75-DGP-13, April 1975.
- Morgan, D. T., Davis, J. P., "High Efficiency Gas Turbine/Organic Rankine Cycle Combined Power Plant", ASME 74-GT-35, April 1974.
- Steinlicht, B., "Low Level Heat Recovery Takes on Added Meaning as Fuel Costs Justify Investment", Power, April 1975.
- Barber, R. E., "Rankine Cycle Systems for Waste Heat Recovery", Chemical Engineering, November 25, 1974.
- Barber, R. E., "Solar Powered Organic Rankine Cycle Engines-Characteristics and Costs", 11th IECEC Proceedings, Sept., 1976-769200.
- 8. Lewis, G. P., Smith, R. D., Harkness, W. P., and Yamada, M., "Sulfuric Acid Plant Rankine Cycle Waste Heat Recovery", 11th IECEC Proceedings, Sept., 1976-769209.
- DeGarmo, E. P. and Canada, J. R., "Engineering Economy", 5th Edition, The MacMillan Company, New York, 1973.
- Popper, H., "Modern Cost Engineering Techniques", 1st Edition, McGraw-Hill Book Company, New York, 1970.
- 11. Oil/Gas Complex Conceptual Design/Economic Analysis, The Ralph M. Parsons Company.
- Purcupile, J. C., "Energy Conservation in Coal Conversion", Carnegie-Mellon University, Pittsburgh, PA, 15213, August 1977.
- 13. Private communication with Ralph M. Parsons Company, February 1978.
- Hittman Associates, Inc., "Assessment of the Rankine Cycle for Potential Application to Solar-Powered Cooling of Buildings", NTIS, August 1974.
- 15. Private communication with Mr. Michael Santucci of Sundstrand Corporation, Rockford, Illinois, February 1978.

- 16. Private communication with V. T. Morgon of Thermo-Electron, Waltham, Massachusetts, February 1978.
- 17. Private communication with J. C. Devol of AFI Energy Systems, Livingston, New Jersey, April 1978.
- 18. "The 3,800 KW FRON Turbine Power Generating System for Mizushima Plant of Mitsubishi Gas Chemical Company, Inc.", IHI Heavy Industries Company, Ltd., Tokyo, Japan, 1975.

Appendix A

Sample Calculations

Rankine Cycle Power Output

The High pressure Separator vapor air condenser (12-1305) is used as an example in these calculations to illustrate the method used in determining possible power output of the ORC.

Data

Heat Source Temp: 300°F

Heat Transfer:

123.3 X 10⁶ BTU/Hr.

Mass Flow:

605,700 Lbm/Hr.

Assumptions

1. Boiler, regeneration and condenser have an effectiveness of 80%.

Sample Calculations

The heat transfer in the boiler is 123.3 X 106 BTU/Hr.

$$QB = 123.3 \times 10^6 BTU/Hr.$$

Assuming boiler effectiveness of 80% the maximum cycle temperature is about 270°F.

$$Tmax = 270$$
°F.

Using the generalized curve of Figure #7, the Rankine cycle efficiency is 15%.

$$y = 15%$$

Multiplying the heat source from the boiler with the cycle efficiency

gives the power output of the cycle.

$$P = yQ_B = .15 \times 123.3 \times 10^6$$

= 18.5 \times 10^6 BTU/Hr.

$$= 18.5 \times 10^6 \text{ BTU/Hr}.$$

- = 5,420 KW
- = 7,275 Hp

Appendix B

1. Cost Analysis

The replacement rankine cycle for the air cooled system (12-1305) is used as an example for the cost analysis presented in this paper.

The installed cost of replacement rankine systems is estimated from the curves of Figure 8 using the estimated power output and cycle temperature calculated in Appendix A.

$$P_{\text{out}} = 5,400 \text{ KW}$$

$$T_{\text{max}} = 270^{\circ}F.$$

From Figure 8 the installed cost is found to be \$500/KW

$$IC = $500/KW$$

Since the total output possible is 5,400 KW the total installed cost is easily found.

$$(IC)_{T} = $500/KW \times 5400$$

= \$2,700,000

Additional cost resulting from enlarging cooling tower capacity is estimated from data given in Reference 10.

Heat exchanger costs were obtained from Reference 13.

$$Hx Costs = $540,350$$

The net cost is found by adding the ORC installed cost plus the cooling tower costs minus the heat exchanger costs.

or \$415/KW

2. DCF - Sensitivity Analysis

A discounted cash flow analysis was performed for various capital investments and rates of electricity. The following is the assumptions used in constructing the curves of Figures 9 and 10.

All Curves

- 1. 20 year project life
- 2. Double-Declining Balance Depreciation
- 3. 48% federal income tax
- 4. \$.003/KW-hr operation and maintenance costs
- 5. 8400 Hrs/year operating

Curve A - Pessimistic outlook

- Capital Cost based on 100% increase of Fig. 8 with credit taken for replacement Heat Exchanger
- 2. Exporting rate for electric power is \$.01/KW-hr escalating at a rate of 6% per year for 10 years

Curve B - Optimistic Outlook

- Capital Cost based on Fig. 8 with credit taken for replacement Heat Exchanger
- 2. Exporting rate for electric power is \$.025/KW-hr escalating at a rate of 8%/year for 10 years

Curve C

- Capital Costs based on Fig. 8 with credit taken for replacement exchanger
- Exporting rate of \$.01/KW-hr escalating at a rate of 6%/year for 10 years

Curve D

- Capital Costs based on 100% increase of Fig. 8 with credit taken for replacement exchanger
- Exporting rate of \$.025/KW-hr escalating at a rate of 8%/year for 10 years.

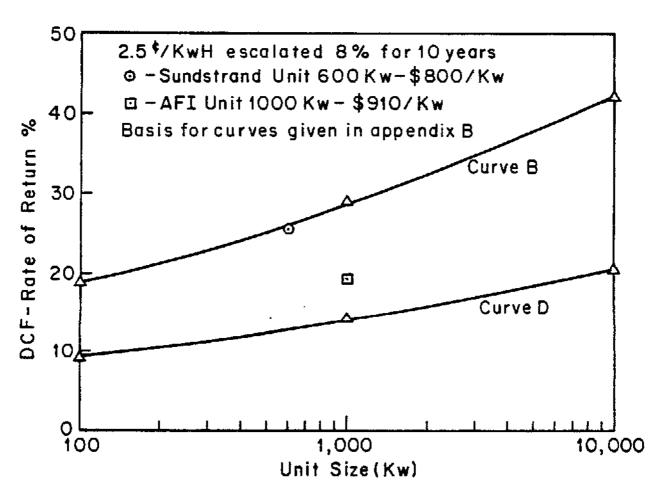


FIGURE 9. RATE OF RETURN VS CYCLE SIZE

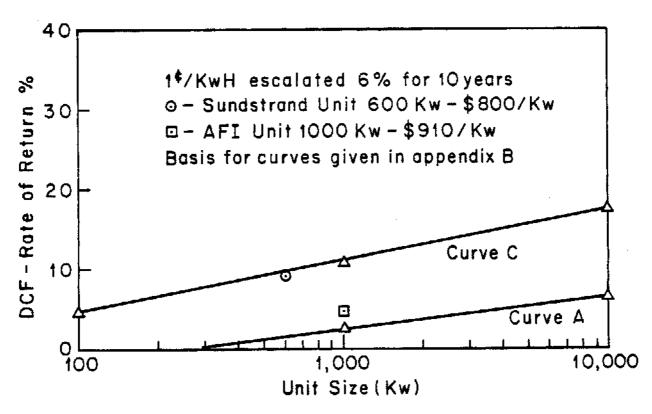


FIGURE 10. RATE OF RETURN VS CYCLE SIZE